

ON THE INTERACTION OF WATER DROPLET WITH A SHOCK WAVE: EXPERIMENT AND NUMERICAL SIMULATION

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The work deals with experimental and computational studies of the water droplets behavior in the flow behind the passing shock wave, and the verification of conducted calculations. In the experiments, a high-speed visualization of the water droplet interaction with the flow behind the passing shock wave was obtained at the flow Mach numbers equal to 0.17 – 0.46 and Weber numbers ranged from 200 to 2200. The calculations simulate conditions of the conducted experiments and are based on the use of the VOF method to resolve the phase interface, LES model to describe turbulence, as well as adapted dynamic grid technology. The article focuses on the study of the flow structure near the droplet and in its wake, the features of the droplet streamlining, the shape evolution type, and the nature of the mass entrainment. Comparison of simulation results with experimental data has shown good agreement with respect to main integral characteristics of the process, i.e. morphology, dynamics, and induction time of droplet breakup.

Keywords: the aerodynamic breakup of drops, shock waves, experiment, mathematical simulation, VOF, LES.

1. Introduction

Aerodynamic crushing of droplets is widely used in various industries, such as energy, aircraft, and rocket engine building, hardening of materials and coatings, chemical industry, etc. Aerodynamic mechanisms are most effective in flows with high parametric gradients, while the interaction of droplet with surging shocks and passing shock waves are the limiting cases of gradient flows. In man-made systems, the interaction of droplets with shock waves takes place, for example, in close proximity to airframe components of the supersonic aircraft [1]. Thunderstorm processes in the precipitation zone, causing large-scale changes in the atmosphere, are also accompanied by the propagation of shock waves in the gas-droplet medium.

The study of droplets dynamics and breakup in the flow behind passing shock waves holds a special place in the subject of the aerodynamic crushing of liquids [2, 3], and since the early works, has developed in two directions. First, this issue concerns the problems related to the combustion of hydrocarbon fuel aerosols in relation to industrial explosion safety [4-6] and to advanced Pulse Detonation Engines (PDE) [7]. The process takes place in two stages: first, droplets crushing behind the shock wave; and second, ignition of the mixture and the movement of the flame front over the spray. The risk of explosion increases in the case of supersonic flow behind the shock wave front. Here, the conditions arise that accelerate the self-ignition of the spray – the increase in gas pressure and temperature during flow deceleration in the tight space created by the suspended matter [8].

Since spraying proceeds equally for all low-viscosity liquids [2, 3, 9], it can be studied with regard to water that as noted is of independent interest. Thus, the second direction appeared in the study of droplet behavior in shock waves, namely dynamics [10-13] and droplets crushing mechanisms at a sudden penetration into the flow [2, 9, 14, 15]. This became the most popular formulation of the problem from the experimental standpoint. This is due to the fact that one can accurately determine the point in time when the shock wave hits the droplet. At that, the visualization of the droplet deformation and the type of its breakup are the most informative to analyze the process itself. There are also hidden processes that are not observable. This concerns the liquid motion inside the droplet during its deformation, the development of the conjugated boundary layer, and the instability of the surface. However, all the ideas about the droplet breakup mechanisms are based only on the data concerning the nature of its deformation before mass entrainment [2, 3, 9]. The movement of the liquid boundary relative to the initial surface of the droplet due to the changing distribution of external pressure gives rise to the most bizarre shapes in different streamlining regimes. Attempts to establish a relationship between the droplet deformation and the external flow pattern were made in early works [11], though, based on an essential simplification of the droplet shape considered as an ellipsoid of rotation with a small axis along the velocity vector. This type of deformation is possible for raindrop type regimes, i.e. freely falling droplet with a diameter of $d_0 \sim 2-3$ mm and steady velocity of $\sim 4-5$ m/s. However, the deformation type in the flow behind the shock wave is quite different, but the model of the conjugated boundary layer formation in the liquid, proposed in [11], is still relevant for liquid jets, films, and droplets [9].

Thus, crushing mechanisms are determined by processes on both sides of the phase interface. Such problems with the small scale resolution of the process are available only for numerical methods, and this is an important aspect to motivate computer simulation of the process. Verification of calculations based on experimental data will allow debugging the numerical

technique and receiving the new data, which are inaccessible in experiments. An important task of numerical simulation, for both verification and research purposes, is the droplet morphology as a response of the liquid sphere to the change in the pressure distribution over the surface during restructuring the field of streamlining of the body with changing shape.

Returning back to the experiments, we should note that the possibility of detecting a droplet with reference to the moment of entering the flow behind the shock wave front, made shock tubes the most effective tool for this kind of research. That is why the experiments with droplets in shock waves were repeatedly reproduced in many laboratories around the world for a wide range of regimes that increase the reliability of the data collected in a number of reviews [9, 15]. There are also unsolved problems associated with the development of innovative technologies and the emergence of advanced materials with complex rheology. Thus, the interaction of droplets with shock waves is one fundamental problem of physical gas dynamics within the framework of the heat and mass transfer in non-equilibrium heterogeneous systems of technogenic and natural origin, while computer simulation of these processes is the relevant and promising research trend.

2. Physical mechanisms of the droplet breakup in shock wave

Aerodynamic crushing modes of low-viscosity liquid droplets are determined depending on the Weber criterion: the We number is the ratio of the disturbing flow force $\sim \rho u^2 d^2$ and the stabilizing surface tension force $\sim \sigma d$, i.e. $We = \rho u^2 d / \sigma$ [2, 9], where ρ and u are the density and velocity of the gas and droplets, while d and σ are the droplet size and surface tension. The following six droplet breakup modes are distinguished according to [2]: vibrational breakup ($8 < We < 12$) (1), bag breakup ($12 < We < 50$) (2), bag-and-stamen breakup ($50 < We < 100$) (3), sheet stripping ($100 < We < 250$) (4), wave crest stripping ($We > 250$) (5), and catastrophic breakup ($We > 250$) (6).

Other classifications are also known. They differ in types of breakup and ranges in We numbers [2, 3]. For example, in [14] it is shown that the change of mechanisms 4 and 5 in shock wave occurs at $We \sim 1000$, rather than $We > 250$. The nominal differences between the classifications would not be important if they were not related to the relevant physical mechanisms. The point is that the traditional classifications are based on the droplet morphology, but in [9], and later in [15] it is proposed to define the droplet breakup types depending on the following physical mechanisms.

1. Types 1-3 are combined into one Rayleigh-Lamb-Taylor instability mode within the expected range of We numbers, $10 \leq We \leq 40$;

2. Types 4 and 5 represent the sheet stripping from the liquid surface layer occurring according to the boundary layer formation mechanisms (4) and the development of its instability (5) within the expected range of We numbers, $40 \leq We \leq 105$;
3. Type 6 is the catastrophic breakup as a consequence of Rayleigh-Lamb-Taylor instability development within an expected range of We numbers, $10^3 \leq We \leq 10^5$.

Currently, the numerical simulation of these processes is a rather challenging task. Here, the only universal tool is a direct numerical simulation (DNS) with full resolution of the phase interface, but because of the huge computational cost, this approach can be used only for a very narrow class of model problems. Alternatively, a technique based on a combination of the VOF method for resolving the phase interface, LES models for describing turbulent flows, and dynamic grid technology adapted to the phase interface, can be employed as well. This approach is less demanding on computing resources and allows describing the behavior of the mobile phase interface at the main turbulent scales. However, it requires further development and testing for the application in applied problems. The most effective testing of numerical technology is conducting simulations in the same conditions as in the available experiments, and comparing the results by the maximum number of indicators, such as the following integral quantitative parameters:

- *droplet deformation rate [10, 12];*
- *the breakup induction period [14].*

This may be the structural characteristics of the process:

- *the deformation scenario and peculiarities of droplet streamlining;*
- *the type of instability of the droplet surface and the nature of the mass entrainment [14].*

The numerical simulation technique, verified by such a set of characteristics, will allow using further calculations for a wide range of tasks and obtaining new data, which cannot be generated through experiments.

3. Experiment statement and preliminary observational analysis

The experiments were carried out using the UT-4M shock tube available at the Institute of Theoretic and Applied Mechanics of the Siberian Branch, Russian Academy of Sciences, which is described in detail in [3, 5]. The installation is characterized by the following main features: the high-pressure chamber and the low-pressure channel are separated by an electro-valve to start the installation by means of a signal from the synchronization system; the low-pressure channel 5 m long allows obtaining a quasi-stationary flow behind the shock wave front during $\sim 500\text{-}600 \mu\text{s}$; the measuring section has a cross-section of 52x52 mm, equipped with quartz windows 20x200 mm in size to carry out shadow visualization, as well as a special device for entering freely falling droplets. The installation is started by the synchronization system when a droplet

enters the measuring section. The same device starts a registration system with the arrival of the shock wave front. Registration is carried out by a multi-frame shadow system with a laser stroboscopic light source. The interval between frames is $30 \pm 0.1 \mu\text{s}$ and the exposure of 30 ns is set by the light source. The spatial separation of the frames on the film is performed by the LFR-3 triggered camera.

The parameters of the shock wave were calculated by the ideal theory of shock waves [16,17], which gives high accuracy for Mach numbers of the shock wave $M_S = V_S/c \leq 4$, where V_S is the velocity of the shock wave front, c is the sound velocity in front of it. The V_S value was measured twice: in experiments conducted to synchronize the installation, V_S was determined by the run time between the pressure sensors with an accuracy of 2 %; when processing shadow images, V_S was determined with an accuracy of 1 % by the position of the wavefront on adjacent frames with a known interval between them.

In the experiments, the interaction of water droplets with shock wave was modeled at We numbers ranged from 200 to 2200. Figure 1 shows a series of process shadow images at $M_S = 1.32$, $We = 2 \cdot 10^3$, $d_0 = 2.81 \text{ mm}$. Shock wave moves from left to right, the front of the shock wave is visible on frame No. 1. Flow parameters behind the wavefront: temperature $T_2 = 354 \text{ K}$ (81 C), density $\rho_2 = 2.0 \text{ kg/m}^3$, and gas velocity $u_2 = 162 \text{ m/s}$.

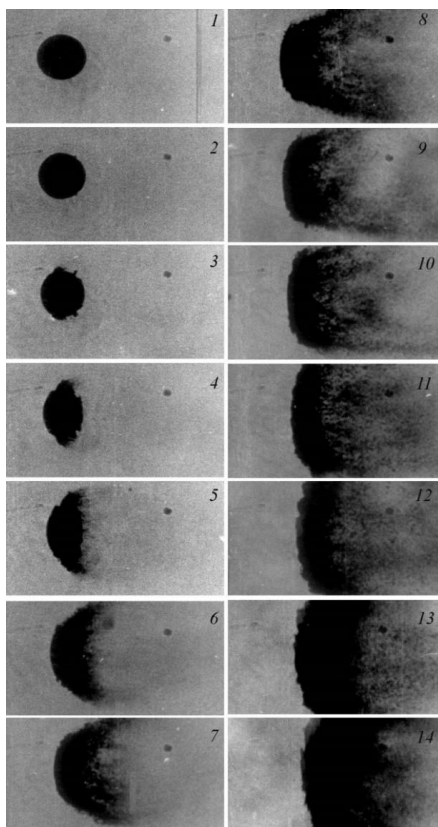


Fig. 1. Shadow images of the water droplet interaction with passing shock wave;
 $We = 2 \cdot 10^3$, frame spacing $30 \cdot 10^{-6} \text{ s}$

In [1] it was proposed to describe the droplet dynamics in shock wave in terms of dimensionless quantities $X = x / d_0$ and $T = t / t_0$, where x is the longitudinal coordinate, t is the physical time, $t_0 = (d_0 / u_2) \cdot (\rho_1 / \rho_2)^{0.5}$. According to numerous experiments, a universal dependence for the induction period of the mass entrainment was found, $t_i / t_0 \approx 0.35-0.4$. It was also argued that up to $T \approx 6-8$ the acceleration of the droplet is constant, $a \approx \text{const}$; then $X \approx kT^2$. Note that this dependence does not refer to the mass center of the droplet, but to the leading edge. Moreover, this dependence is quite approximate at the ambiguous influence of the midsection growth and the mass entrainment. Because of this fact, the value of k differs markedly, according to the results of different authors. Thus, according to [1] $k = 0.71-1.1$, according to [13] $k = 0.8 - 1.1$. As shown in [10, 12], the droplet dynamics are much more complicated, the acceleration is not constant, and decreases sharply with the development of intense mass entrainment within the range $T \approx 1-3$.

However, for a preliminary analysis of these experiments, we will also use estimates expressed in dimensionless parameters. For conditions corresponding to Fig.1, $t_0 = 387 \mu\text{s}$, taking into account the residence time of the droplet in the flow corresponding to frame No. 1 ($22 \cdot 10^{-6}$ s), the total observation time of the droplet $t_N = 412 \mu\text{s}$ ($t_N / t_0 = 1.05$). During this time the droplet has moved by the distance $x = 2.79$ mm ($x / d_0 = 0.99$), the coefficient $k = 0.9$. Beginning of mass entrainment is seen on frame No. 5, the mass entrainment induction period is $t_i = 142 \pm 15 \mu\text{s}$, while in dimensionless form, $t_i / t_0 = 0.36$ that, similarly to coefficient k , is close to the accepted data on the interaction of the droplets with the shock wave.

The results of experiments conducted according to this technique are published in [2, 4, 7, 8, 10]. Analysis of the above experiment, as well as other experiments in this series, shows that in the accepted terms, all modes were implemented in the right conditions. They can serve a basis for verification of numerical technology in terms of both quantitative data, and, most importantly, graphical data on the morphology of droplet interacting with a shock wave. To date, these data are the most detailed of those obtained within the range of We numbers 200 - 2200, known in the literature.

4. A mathematical model to calculate the droplet breakup in the flow beyond the shock wave

At present, a wide range of mathematical models, for example [18-20], which differ in the ways of describing the gas and particles motion processes, as well as used assumptions and restrictions, is proposed to describe gas-droplet flows. The choice of a suitable model is carried out, as a rule, taking into account the available information about the flow structure and the

required accuracy of its description. The most complete reviews on methods for solving problems with moving contact boundaries are presented in [21-23].

Among the algorithms based on continuous volume markers, the most popular method due to the ease of implementation and efficiency is the volume of fluid (VOF) method [23], which proved to be good when calculating free surface flow. The idea of the method is that liquid and gas are considered as a single two-component medium, in which the spatial distribution of phases within the computational domain is determined by a special marker function $F(x, y, z, t)$. The volume fraction of the liquid phase in the cell under consideration is taken as follows: $F(x,y,z,t) = 0$, if the cell is empty, $F(x,y,z,t) = 1$, if the cell is completely filled with liquid, and $0 < F(x,y,z,t) < 1$, if the phase interface is within the cell. Since the free surface moves with the liquid, tracking the movement of the free interface is performed by solving the transfer equation of the liquid volume fraction in the cell:

$$\frac{dF}{dt} + V \cdot \nabla F = 0, \quad (1)$$

where V is the velocity vector of the two-phase medium, found from the solution of a system of hydrodynamics equations, i.e. the mass conservation or continuity equations:

$$\frac{d\rho}{dt} + \nabla(\rho \cdot V) = 0, \quad (2)$$

and motion equations or the momentum conservation law:

$$\frac{d\rho V}{dt} + \nabla(\rho V \times V) = -\nabla p + \nabla(\tau) + F, \quad (3)$$

Here τ is the viscous stress tensor, F is the volumetric force vector, p is the static pressure, ρ is the density of the two-phase medium. Components of viscous stress tensor τ_{ij} are written as:

$$\tau_{ij} = \mu \left(\frac{dU_i}{dx_j} + \frac{dU_j}{dx_i} - \frac{2}{3} \delta_{ij} \frac{dU_k}{dx_k} \right), \quad (4)$$

where μ is the dynamic viscosity of a two-phase medium, U_{ij} are the velocity vector components. The density and molecular viscosity of the considered two-component medium are found through the volume fraction of the liquid in the cell according to the mixture rule:

$$\rho = \rho_1 F + (1 - F) \rho_2, \quad (5)$$

$$\mu = \mu_1 F + (1 - F) \mu_2, \quad (6)$$

where ρ_1 and μ_1 are the density and viscosity of the liquid, ρ_2 and μ_2 are the density and viscosity of the gas. The obtained values of density ρ and viscosity μ are included in the motion equations and determine the physical properties of the two-phase medium.

When considering fluid flows with a free interface, special attention is paid to the surface tension phenomenon. The study of the flows driven by the surface tension forces is a very complex independent task. Therefore, the advantages of the VOF method also include the fact that it allows a relatively simple simulation of the effect of surface tension forces. Most often, the CSF algorithm [24] is used to simulate the surface tension in the framework of the VOF method, which involves the introduction of an additional volumetric force, F_s into the motion equations, which is determined from the correlation:

$$F_s = \sigma k \nabla F, \quad (7)$$

where σ is the surface tension coefficient, k is the curvature of the free surface, which is defined as the divergence of the normal vector:

$$k = \nabla \left(\frac{n}{|n|} \right), \quad (8)$$

The normal to the free surface is calculated, in turn, as the gradient of the volume fraction of the liquid phase in the cell:

$$n = \nabla F, \quad (9)$$

In this case, the value of the normal vector on the solid wall is determined by limiting wetting angle θ :

$$n = n_w \cos \theta + \tau_w \sin \theta, \quad (10)$$

where n_w and τ_w are normal and tangential vector components with respect to the wall.

The simulation of turbulence is another important factor when calculating droplet breakup. In this paper, we used the LES model [25] for turbulence modeling, according to which the solution of filtered Navier-Stokes equations is necessary to describe turbulent flows. Then, the above-mentioned system of hydrodynamics equations can be rewritten in the following form:

– continuity equation:

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_i} (\rho \bar{u}_i) = 0, \quad (11)$$

– momentum conservation equations

$$\frac{\partial}{\partial t} (\rho \bar{u}_i) + \frac{\partial}{\partial x_j} (\rho \bar{u}_i \bar{u}_j) = \frac{\partial}{\partial x_j} \left(\mu \frac{\partial \sigma_{ij}}{\partial x_j} \right) - \frac{\partial \bar{\rho}}{\partial x_i} - \frac{\partial \tau_{ij}}{\partial x_j}, \quad (12)$$

where $\sigma_{i,j}$ is the viscous stress tensor, whose components have the form:

$$\sigma_{ij} = \left[\mu \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right) \right] - \frac{2}{3} \mu \frac{\partial \bar{u}_i}{\partial x_i} \delta_{ij}, \quad (13)$$

where μ is the molecular viscosity.

Tensor $\tau_{i,j}$ is called a subgrid stress tensor, and its components are determined by analogy with RANS models from the Boussinesq approximation:

$$\tau_{ij} - \frac{1}{3}\tau_{kk}\delta_{ij} = -2\mu_t \bar{S}_{ij}, \quad (14)$$

Here $\bar{S}_{i,j}$ is strain velocity tensor:

$$\bar{S}_{ij} = \frac{1}{2} \left(\frac{\partial \bar{u}_i}{\partial x_j} + \frac{\partial \bar{u}_j}{\partial x_i} \right), \quad (15)$$

The value μ_t is called the subgrid viscosity. In this paper we used the subgrid viscosity model proposed by Smagorinsky:

$$\mu_t = \rho L_s^2 \dot{\gamma}, \quad |\bar{S}| \equiv \sqrt{2\bar{S}_{ij}\bar{S}_{ij}}, \quad (16)$$

where L_s is mixing length of subgrid scales: $L_s = \min(kd, C_s V^{1/3})$

Here k is Karman constant, d is the distance to the nearest wall, V is the volume of the computational cell, C_s is the Smagorinsky constant. In the present paper $C_s = 0.17$.

The method of solving equations (1)-(10) and the main features of numerical studies are described in [26-28]. The difference analog of the convective-diffusion equations was found using the finite volume method for structured multi-block grids, whose application ensured the persistence of the obtained scheme. To approximate convective terms of the hydrodynamics equations (3), the central difference scheme of second order was used. An implicit first-order scheme was employed to approximate the unsteady terms of the hydrodynamic equations. Diffusion fluxes and source terms were approximated with a second order of accuracy. The connection between the velocity and pressure fields was realized by SIMPLEC procedure on the combined grids. This approach made it possible to overcome the difficulties described above associated with the resolution of the mobile phase interface. The resulting system of finite-difference equation was solved by an iterative method using a multigrid solver.

Using the described technique, the simulation of water droplet breakup in the flow behind the shock wave was carried out for different Weber numbers within the range of $200 \leq We \leq 2200$. The calculated area was a parallelepiped with dimensions of $3 \times 3 \times 5$ cm. The inlet condition with fixed velocity was set on one of the faces of the parallelepiped, determined from the Weber number. On the other faces of the computational domain, free exit conditions were set. At the initial time point, a spherical water droplet with a diameter $d_0 \sim 2-3$ mm was placed at a distance of 5 mm from the inlet to the computation domain. The droplet was affected by the passing shock

wave generating the air flow. The Cartesian uniform computational grid was used in the calculations. The total number of grid nodes was 6.5 million points. However, methodical calculations have shown that such a grid is not enough to resolve the phase interface of the formed small droplets. Therefore, the gradient adaptation technology of the computational grid was applied. With this technology, in the course of calculations, the grid is automatically concentrated in the area of large solution gradients. The gradient of the liquid volume fraction was chosen as the control parameter. An example of how the original grid changes during gradient adaptation for a flow velocity of 60 m/s is shown in Fig. 2. In the field of high gradients, computation cells are four times smaller than those in the initial grid. In the end, the total number of computational nodes of the optimized grid in the course of calculation was approaching 15 million. The use of such a detailed grid made it possible to resolve secondary droplets up to 20 microns in size.

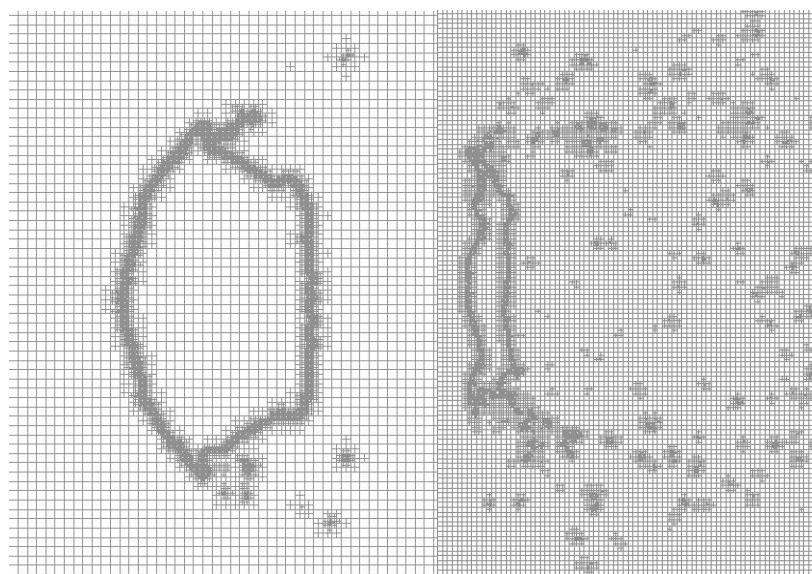


Fig. 2. Evolution of the computational grid near the phase interface by means of gradient adaptation.

4. Simulation results and comparison with experiments

The following physical properties of the phases were used in the calculations: water density and viscosity 998.2 kg/m^3 and $1.003 \cdot 10^{-3} \text{ kg/m}\cdot\text{s}$, surface tension coefficient $\sigma = 0.073 \text{ N/m}$; and air viscosity $1.789 \cdot 10^{-5} \text{ kg/m}\cdot\text{s}$. To simulate the flow regimes within the range $200 \leq We \leq 2200$, the flow velocity varied from 60 to 170 m/s. The ideal gas model was used.

Figure 3 shows an example of simulating the behavior of a water drop in the flow at $We = 208$, the Mach number of the shock wave $Ms = 1.109$, drop size $d_0 = 2.73 \text{ mm}$, the gas velocity $u_2 = 60 \text{ m/s}$, the density $\rho_2 = 1.53 \text{ kg/m}^3$, and the time constant $t_0 = (d_0 / u_2) \cdot (\rho_1 / \rho_2)^{0.5} \approx 1170 \text{ }\mu\text{s}$. Series in Fig. 3a corresponds to the period of drop's stay behind the shock wave $t_N = 520 \text{ }\mu\text{s}$, as it was in the experiment, while Fig. 3b shows the possible

evolution of the drop during the time up to $900 \mu\text{s}$ as if the quasi-stationary flow behind the shock wave existed so long.

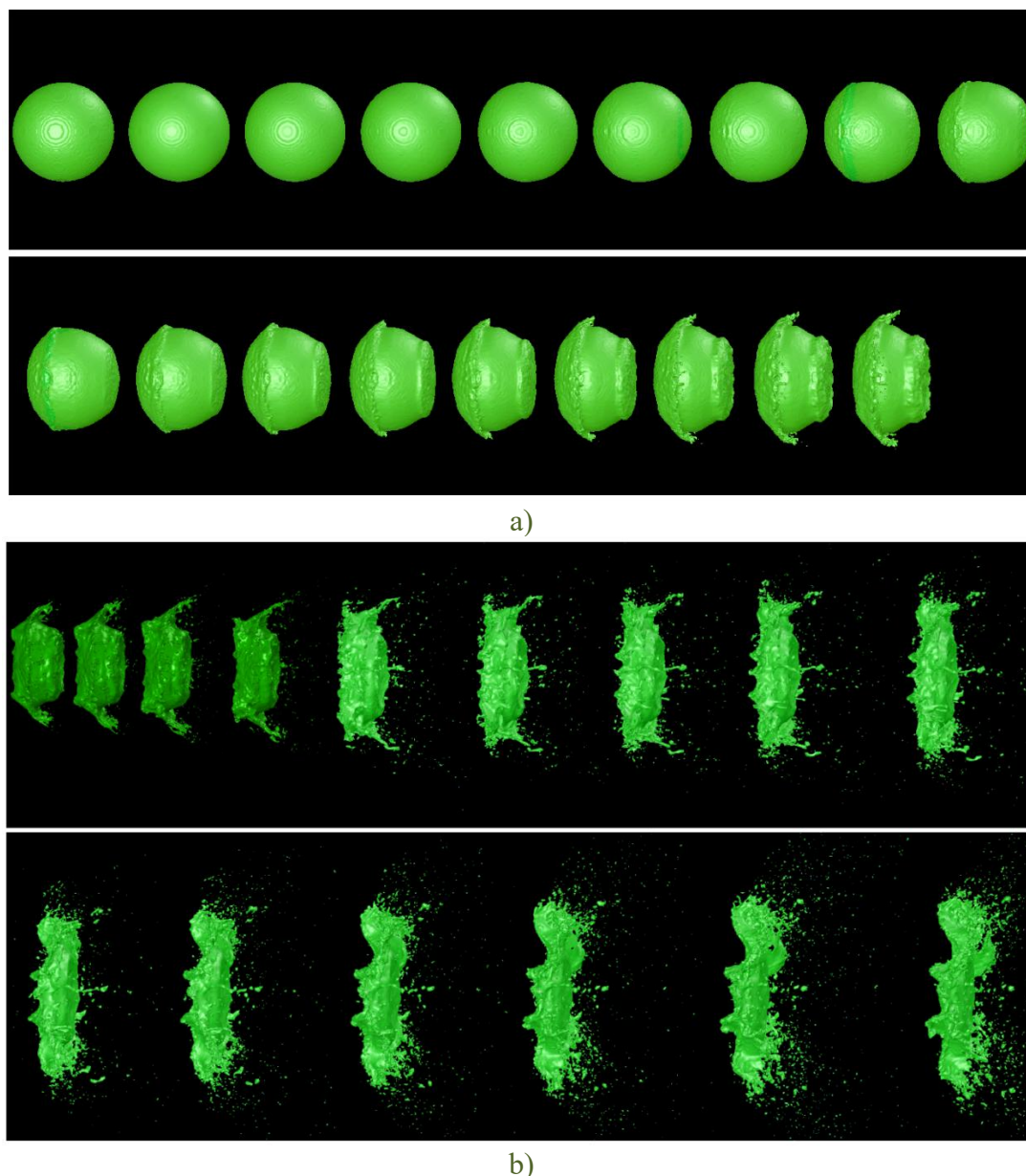


Fig. 3. Numerical simulation of the water droplet behavior behind the shock wave at $We=208$;
a) period up to $520 \mu\text{s}$, b) period of $550-900 \mu\text{s}$.

A comparison of the calculation with the experiment at similar points in time is shown in Fig. 4. As can be seen, there is a good qualitative agreement of the data, both in terms of the droplet shape and its deformation dynamics. In addition to the external similarity of the droplet shape, there is a quantitative index of deformation that affects its dynamics, namely, the growth of the droplet cross-section. The dynamics of the droplet midsection growth (d / d_0) is shown in Fig. 5, where a satisfactory agreement of the mean deformation rate in the experiment and calculations is shown.

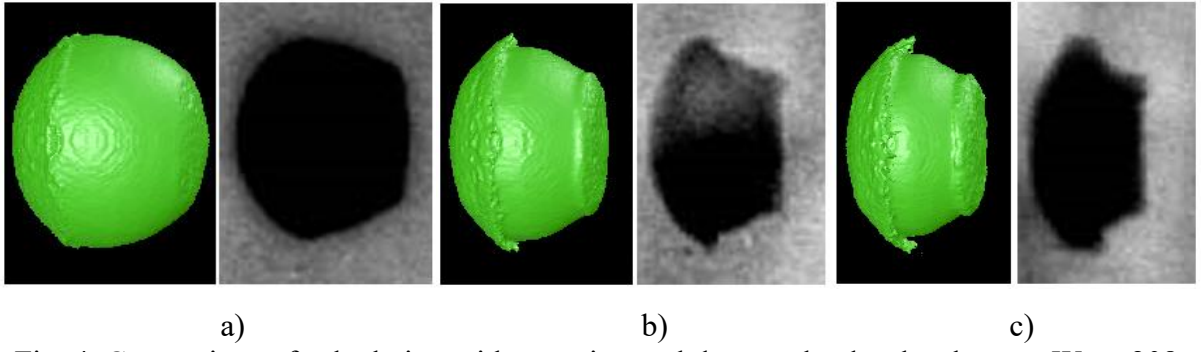


Fig. 4. Comparison of calculation with experimental data on the droplet shape at $We = 208$; a) $240 \mu s$; b) $360 \mu s$; c) $420 \mu s$.

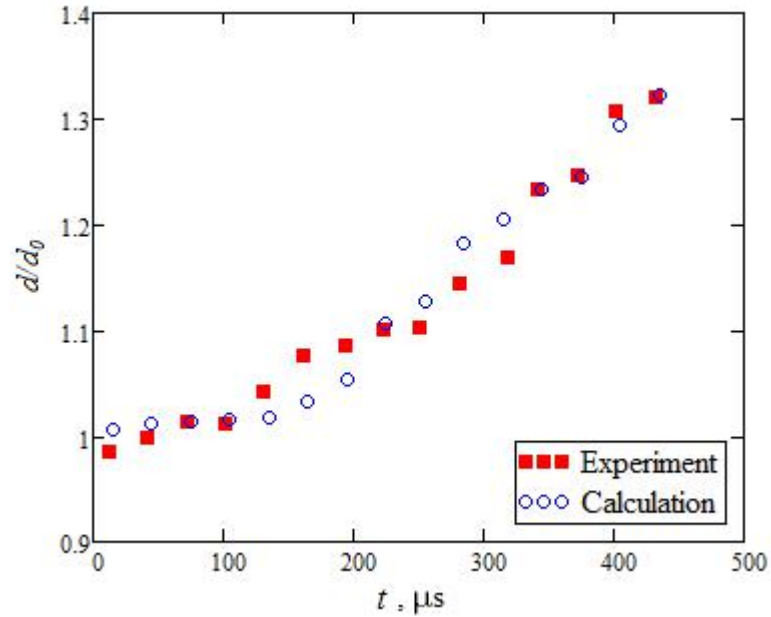


Fig. 5. The transverse deformation rate of a water drop at $We = 208$.

Figure 6 shows the simulation result of water droplets interaction with shock wave at $We = 360$, $Ms = 1.144$, droplet size $d_0 = 2.7 \text{ mm}$, gas velocity $u_2 = 77 \text{ m/s}$, density $\rho_2 = 1.53 \text{ kg/m}^3$, time constant $t_0 \approx 865 \mu s$, the droplet residence time in the flow $t_N = 440 \mu s$ ($T_N = t_N / t_0 = 0.47$). The beginning of mass entrainment is observed on frames No. 9-10 both in the experiment and the calculations (Fig. 7, b), the breakup induction period $t_i \approx 300 \text{ mks}$, $T_i = t_i / t_0 = 0.35$.

As can be seen, there is a good qualitative agreement between the numerical simulation and the experimental data at similar time points, but one should not expect complete coincidence of calculations with experiment at a late stage of the process, shown in Fig. 7c. The point is that in the experiment there are circumstantial factors that are difficult to take into account in the calculations. First of all, this concerns the nonsphericity of the droplet due to fluctuations during the separation from the capillary. Here the general picture and rate of deformation are more important since they influence the induction period and the nature of the mass entrainment. As is

seen from Fig. 8, the calculated data well describe not only the average deformation rate but also the wave microstructure of the process.

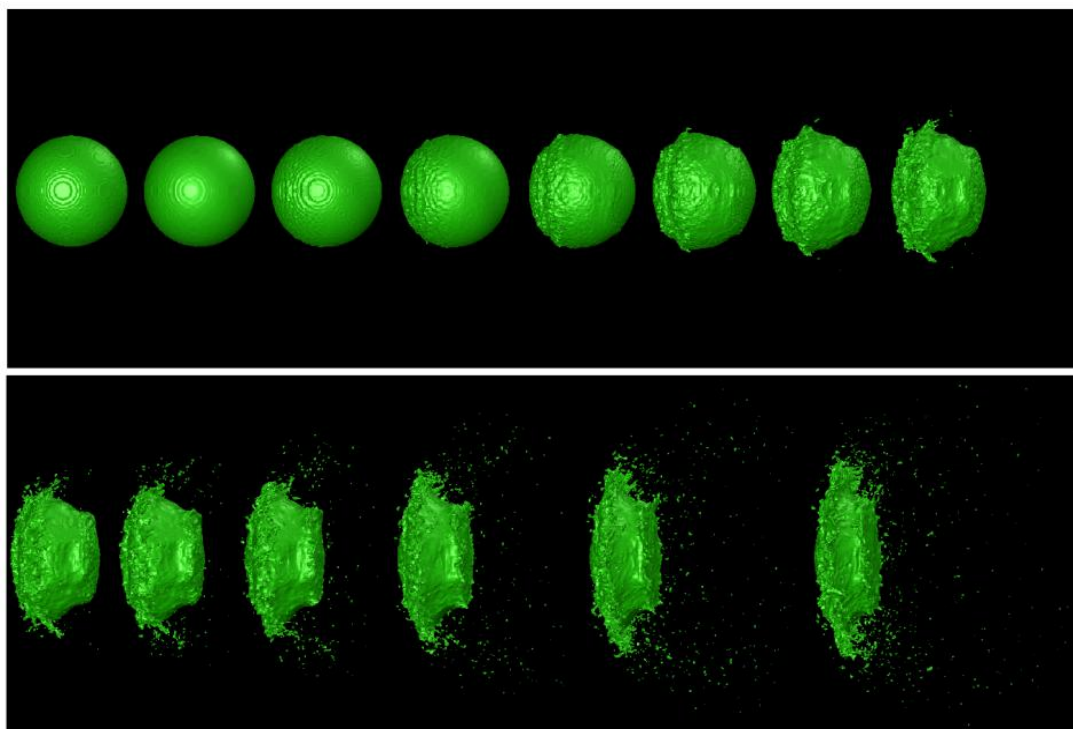


Fig. 6. The breakup of water droplets beyond the shock wave at $Ms = 1.144$, $We = 360$.

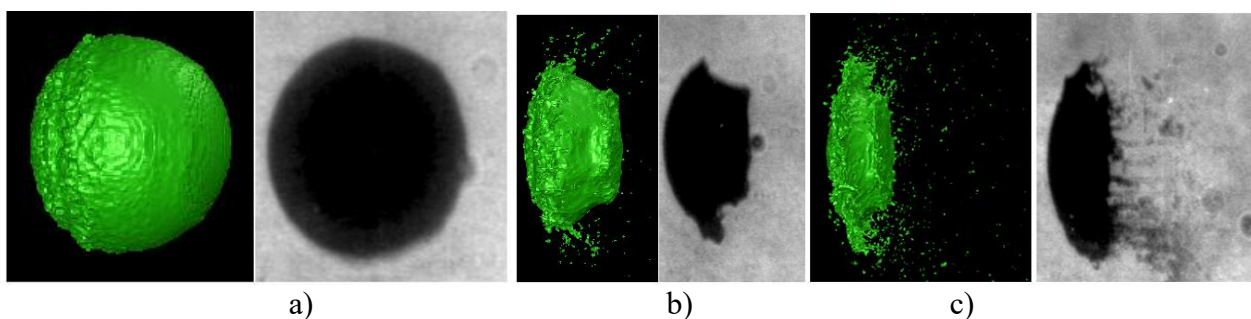


Fig. 7. Comparison of calculation results with experimental data for $We = 360$ at the following time points: $120 \mu s$ (a), $300 \mu s$ (b), $390 \mu s$ (c).

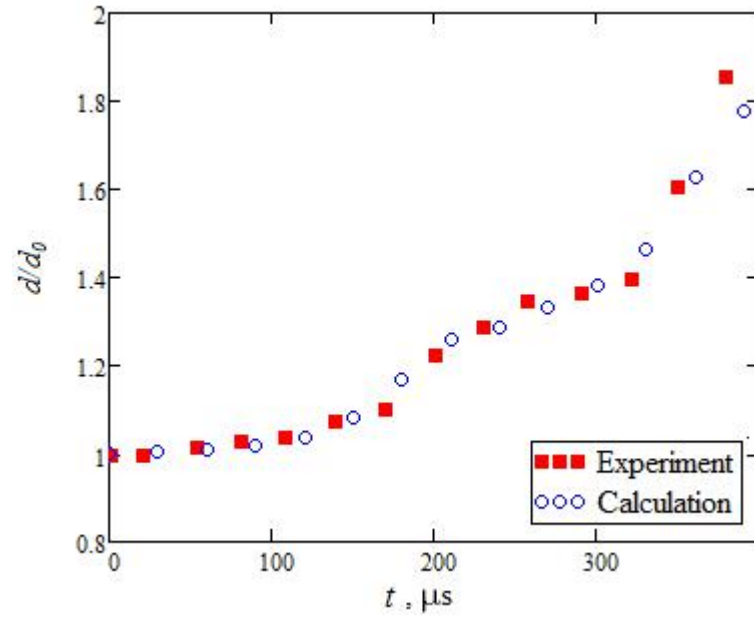


Fig. 8. The water droplet deformation rate at $We = 360$.

The simulation results of the water droplets interaction with shock wave at $We = 650$ are shown in Fig. 9. For a droplet with $d_0 = 2.73$ mm, this is realized at a Mach number of the shock wave $Ms = 1.19$, gas velocity behind the shock wave front 101 m/s, density $\rho_2 = 1.71$ kg/m³, the time constant $t_0 = 657$ μs . The observation time at $N = 15$ is $t_N = 435$ μs , while the dimensionless value of $T_N = t_N / t_0 = 0.75$ corresponds to the double period of mass entrainment induction, i.e. the beginning of the mass entrainment should be sought on frames No. 7-8, while the time of breakup induction $t_i \approx 210$ -240 μs , $T_i = t_i / t_0 = 0.35$.

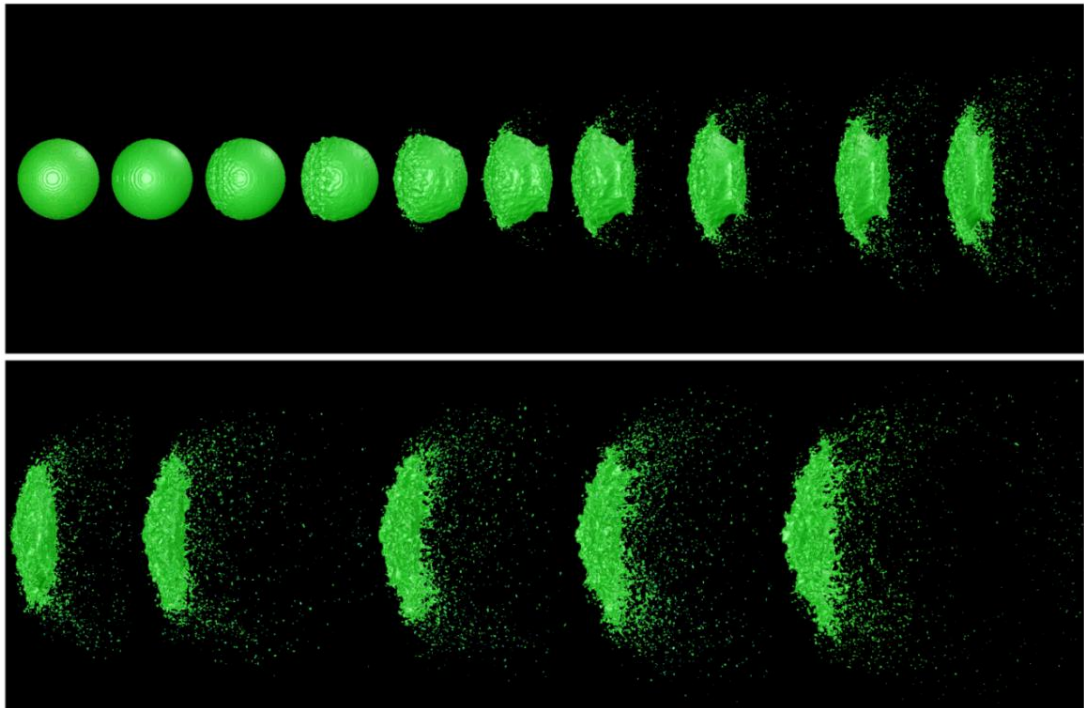


Fig. 9. The behavior of water droplet at interaction with shock wave at $Ms = 1.19$, $We = 650$.

The possibility of observing the later stage of the process ($T_N = 0.75$) allows seeing another feature, namely, the transverse stretching of the film at the flat bottom of the drop on frames No. 7-8, which later becomes the mass entrainment zone (Fig. 10c). This occurs at the $We = 360$, though not so pronounced. At $We = 208$, the limited observation time T_N in the experiments did not allow observing mass entrainment in general, and that in the bottom region of the droplet, in particular. A comparison of the calculation results with the experimental data obtained at the $We = 650$ is shown in Fig. 10, which is indicative of good qualitative agreement between the simulation results and experimental data in terms of droplet shape at the characteristic stages of the process. The growth rate of the transverse size of the droplet in Fig. 11 also shows good quantitative agreement with the experiment. It is important to note that not only the average growth rates are close to each other, but similarly, as at $We = 360$, the phases of surface waves coming to the periphery of the drop coincide. They can be seen by the non-monotonic nature of the growth of the midsection.

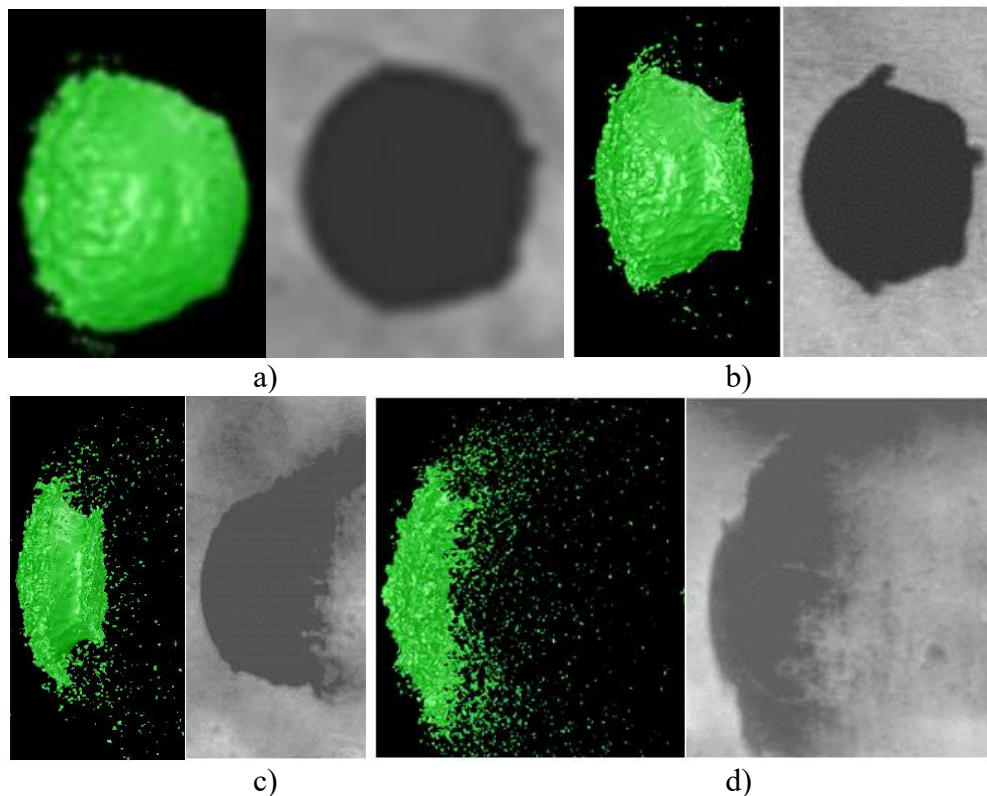


Fig. 10. Comparison of the droplet shape in the calculation and experiment at $We = 650$.
a) $120 \mu\text{s}$; b) $150 \mu\text{s}$; c) $240 \mu\text{s}$; d) $390 \mu\text{s}$.

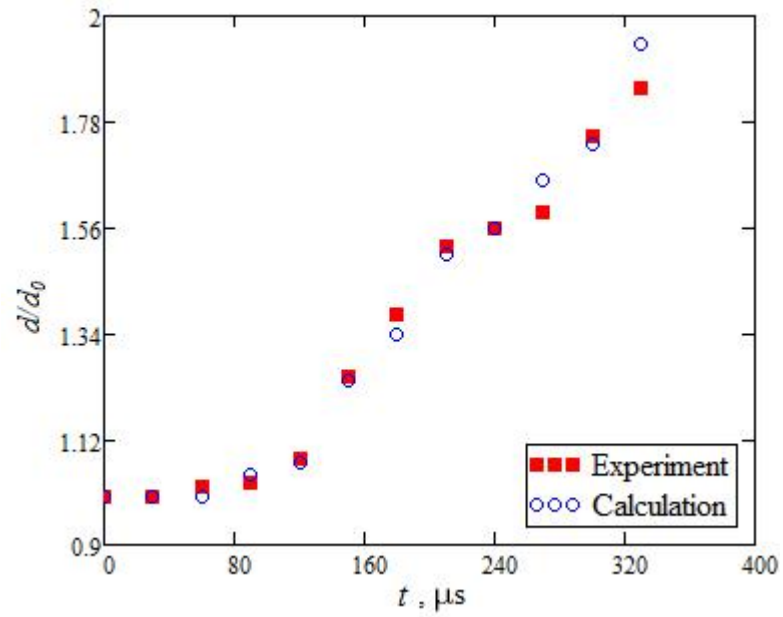


Fig. 11. The deformation rate of water droplet behind the shock wave at $We = 650$.

Figure 12 shows the numerical simulation result of the interaction of water droplet of $d_0 = 2.79$ mm with shock wave at $We = 2260$. This mode is implemented at the Mach number of the shock wave $Ms=1.34$, the flow velocity behind the shock wave front $u_2 = 170$ m/s, and the density $\rho_2 = 2.04$ kg/m³. In this experiment, the time constant $t_0 \approx 362$ μ s, the beginning of the mass entrainment falls within the interval between frames No. 5 and 6, i.e. the induction period $t_i \approx 135$ μ s, $T_i = t_i / t_0 = 0.37$.

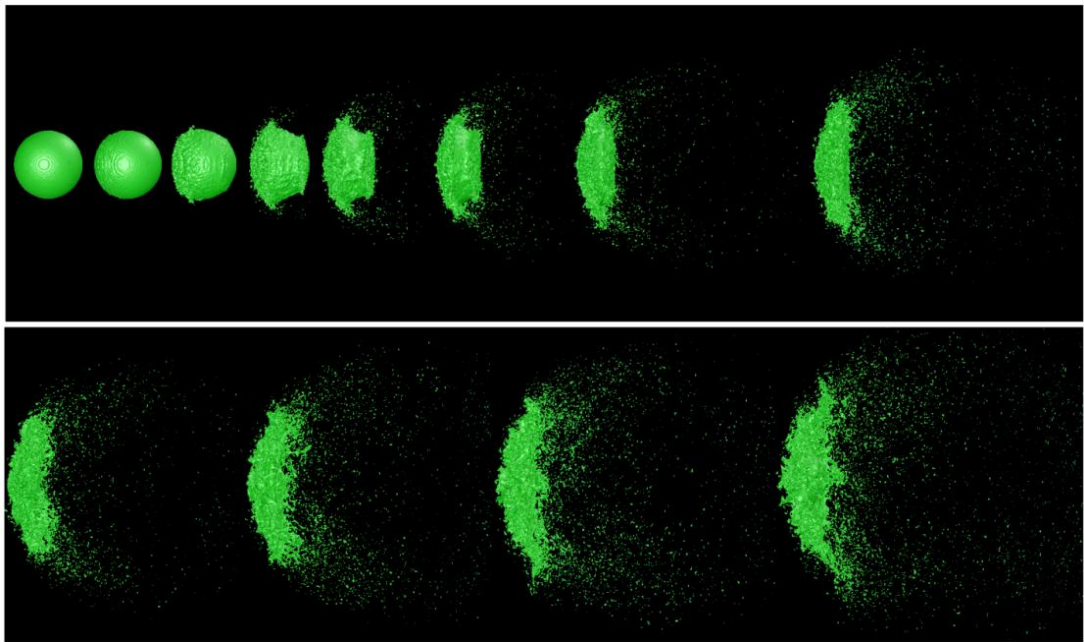


Fig. 12. Water droplet breakup dynamics behind the shock wave at $We = 2260$.

Figure 13 presents a comparison of the calculation with the experiment on droplet shape, which, as is seen, is quite satisfactory. It is also seen that at the beginning of the mass entrainment (Fig. 13b) microdroplets are observed in front of the windward surface of the mother droplet, excluding only the area near the critical point. This fact deserves special attention and will be discussed in the next section since this type of breakup is significantly different from the modes at $We = 360$ and 650 , where stripping of the liquid film occurs in the midsection plane and in the bottom region of the drop.

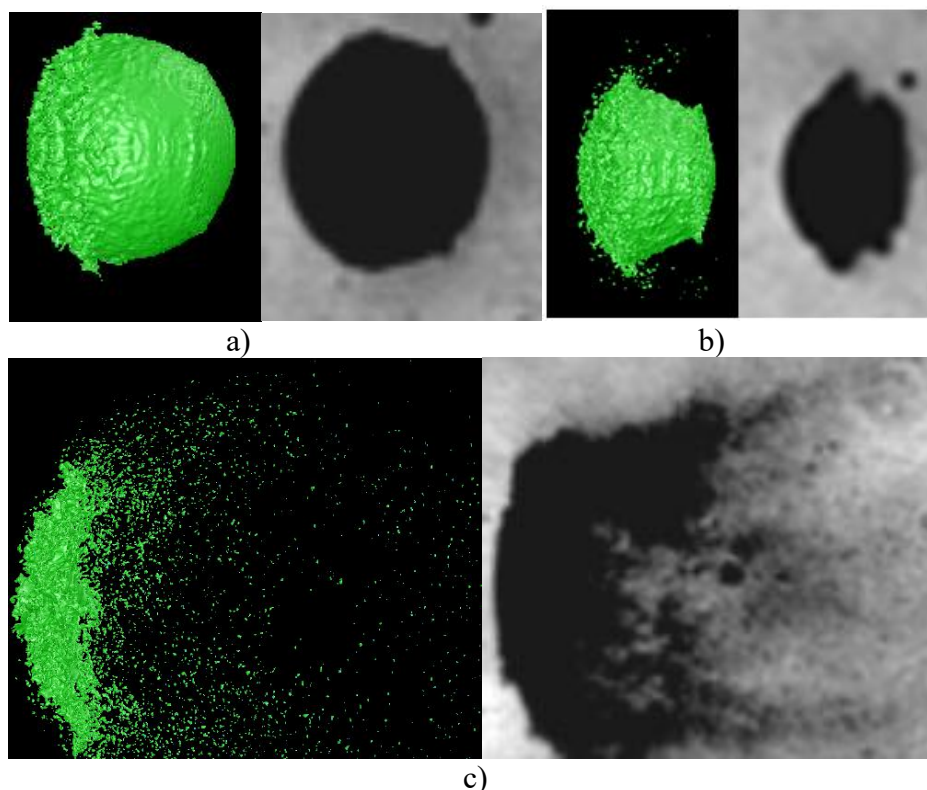


Fig. 13. Comparison of calculation with experiment at $We = 2260$;
a) $60 \mu s$, b) $90 \mu s$, c) $270 \mu s$.

A comparison of the lateral deformation rate of a droplet in the calculations and the experiment in this mode has shown a good agreement up to the interaction time between $100 - 120$ microseconds. At the stage of developed mass entrainment, the uncertainty of the droplet boundary against the background of a dense spray does not allow determining correctly the size of the droplet's integral part in the shadow images obtained in the experiment and verifying the calculations in terms of the deformation rate.

4. Results and discussion

Table 1 shows the gas-dynamic and other characteristics of the regimes considered, two of which deserve special attention. First, it is the mass entrainment induction period t_i , because it is

one of the quantitative indicators to compare calculations with experiments, and secondly, the time constant $t_0 = (d_0 / u_2) \cdot (\rho_1 / \rho_2)^{0.5}$, as a generalized parameter of the interaction mode between the droplet and the flow.

Table 1

We	M _S	u_2 , m/s	ρ_2 , kg/m ³	t_0 , μ s	$T_N = t_N / t_0$	t_i , μ s	$T_i = t_i / t_0$
208	1.109	59.8	1.53	1168	0.4	600	0.5
360	1.144	77.9	1.61	865	0.47	330	0.36
653	1.19	101	1.71	657	0.96	220	0.35
2000	1.32	162	2.0	387	1	142	0.36
2260	1.34	170	2.04	362	1	135	0.37

The peculiarity of the experiments conducted in shock tubes consists in the limitation of the observation time t_N due to the limited duration of the quasi-stationary flow behind the shock wave, in present experiments $t_N \approx 500-600 \mu$ s. The *Table* shows that with the increase in the We number, the dimensionless time $T_N = t_N / t_0$ increases and covers progressively the later stages of the process, but in the mode with $We = 208$, the observation time appeared to be insufficient to detect the droplet breakup. Therefore, in the calculations, the "observation" time was extended to 900μ s (Fig. 3b), the time $t_i \approx 600 \mu$ s was obtained from the calculated "visualization", while the dimensionless time $T_i = t_i / t_0$ was slightly overstated in comparison with other modes. This is due to the fact that the regime with $We \approx 200$ appeared in the transition region between Rayleigh-Taylor instability and stripping mechanisms, while at $We < 200$ the breakup delay increases and reaches the value of $T_i \approx 2$ [29]. The transition region is characterized by the simultaneous development of several breakup mechanisms, and thus the process is called a mixed or multimode regime [29]. In this example, this type of deformation is typical for the entrainment mechanism at an early stage of interaction (Fig. 3a), and the development of Rayleigh-Taylor waves at a late stage (Fig. 3b).

As for the other modes, the calculations have shown a good agreement of the induction period t_i with the experiments. The dimensionless induction time is independent of the We number and equals to $T_i = \text{const} \approx 0.36$. Then, for the estimated delay of the breakup at $We > 250$ one can use the empirical formula $t_i = 0.36 \cdot (d_0 / u_2) \cdot (\rho_1 / \rho_2)^{0.5}$, the velocity u_2 and density ρ_2 of the gas is computed based on Mach number of the shock wave [16,17].

The second quantitative factor to verify the calculation is deformation dynamics. In calculations and experiments, the average growth rates of the droplet midsection are quite close to each other. At that, midsection growth is non-monotonic. In [14] this was explained by the arrival of concentric surface waves to the equator of a droplet, though at $We = 208$ there is a divergence of the phase of the wave (Fig. 5). This may be due to the initial non-sphericity of the droplet at the

time of impact by the shock wave. In other modes, the phases of surface waves in the calculations and experiments coincide with high accuracy (Figs.8 and 11). Thus, the comparison of numerical simulation with the experiment has shown good agreement in terms of the droplet shape, deformation dynamics, and breakup delay that indicates a high-resolution capability of the computational algorithm.

Given the known data on the time of complete breakup of the droplet $t / t_0 = T \approx 5$ [2, 29] we note that in this paper, only the early stage ($T_i < T_N < T$) is considered, though it is very important for understanding the processes occurring inside and outside the droplet. Although the problem of internal processes is not set here, it is useful to highlight the main issues: this concerns "macroscopic" movement of the liquid in the drop during deformation, the development of the conjugated boundary layer, and surface instability. These are hidden processes that are not observable and can be judged only indirectly by the nature of the droplet deformation and the streamlining pattern.

The relationship between the droplet shape and the streamlining pattern is seen in the response of the liquid sphere to the change in the velocity field near it. That is, to understand the evolution of the droplet shape, a general picture of the flow pattern is necessary, but panoramic methods of velocity measurements, such as for example, PIV, are not applicable in shock tubes due to the limitation of response speed. Therefore, in [14] experiments were carried out with the droplet model in a stationary flow with at the Re number $\sim 10^4$, equal to that in a shock wave. The model had the shape typical to one at the stage of initial mass entrainment (Figs. 4b, 7b, 10b, and 13b). The features of the flow around such a body according to PIV data will be considered later, while here we will focus on the results of numerical simulation of the flow around the drop at different stages of its shape evolution.

The flow pattern for the regime represented in Fig. 3 ($We = 208$) is shown in Fig.14. When the droplet is slightly deformed, the flow around the droplet is close to the flow around the sphere at the corresponding Reynolds numbers ($Re \sim 10^4$) with a separation near the midsection (1). Then, 200 μs after a drop enters the flow, a toroidal vortex (2) and a reverse flow (3) are formed simultaneously with the generation of two annular waves (4) and (5) on the initially spherical droplet. This type of deformation is typical for the entire investigated range of We numbers, while the analysis of the gas velocity field, performed in [14], revealed the characteristics of flow around such bodies, which can be traced also in numerical simulation (Fig.14).

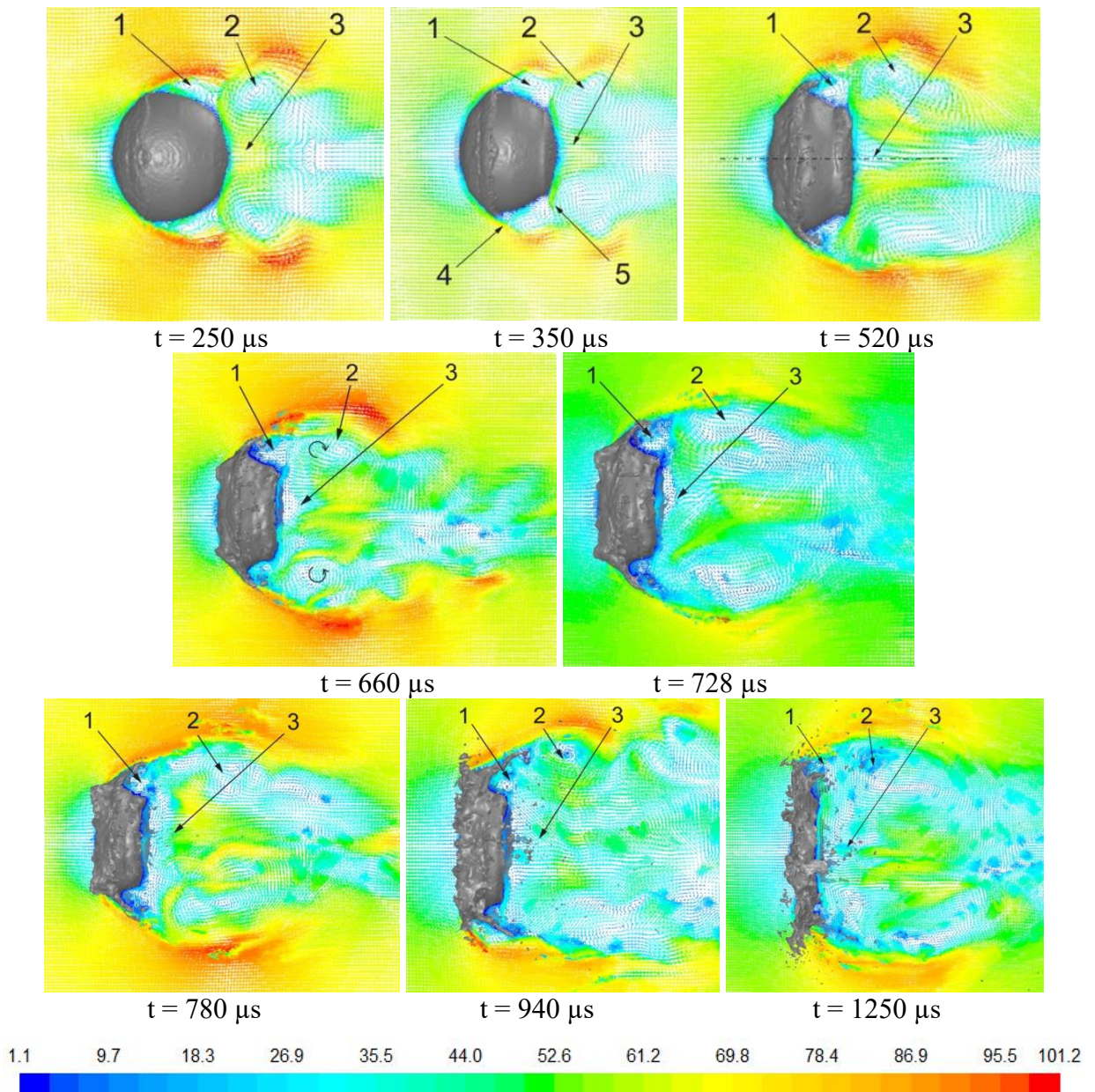


Fig. 14. The flow field near the droplet and in the droplet wake in different time points, $We = 208$;
 1 – toroidal vortex in the flow separation region; 2 – solitary toroidal vortex;
 3 – reversing flow near the wake axis; 4 – kink of the generatrix at the point of midsection growth;
 5 – fracture of the generatrix in the bottom of the drop.

The external flow is unsteady, but it constantly contains several gas-dynamic structures, whose behavior could be interpreted as follows:

- The first higher wave is generated even at the spherical shape of the droplet at the point of flow separation, where its velocity is maximal (red region), while the pressure is minimal. The recirculation zone with a counter-current flow along the droplet surface (area 1) is visible behind the separation point. In the conjugated boundary layer, the liquid moves to the midsection which along with decreased pressure contributes to the growth of the latter.

- The second stationary structure is in the droplet wake and represents a second recirculation zone with intense counter-current flow along the wake axis (area 3). The pressure distribution in the bottom of the droplet caused by this impact flow makes the surface flat, while the radial bottom flow separates at its boundary.
- Here is formed a second fracture of the generatrix, which is associated with the radial spreading of the liquid over the bottom surface. As can be seen in Figs. 6, 9, and 12, this leads to the formation of a liquid disk, whose edge supplies the droplet crushing products to the aerodynamic wake of the drop.
- Spreading radially over the bottom surface, the flow is separated near the second wave. At that, part of it penetrates into the first separation zone (*1*) along the droplet surface, while the rest of the flow enters the third permanent structure (*2*) (Fig.15).
- The structure (*2*) partially separates the first two structures and is an isolated toroidal vortex. Without interacting with the droplet surface, it originates at droplet bottom and drifts into the aerodynamic wake.
- Comparison of shadow images of the droplet at the beginning of the breakup with the external flow field has shown that the mass entrainment occurs from the wave crests, i.e. annular fractures of the generatrix, while they are formed at the flow separation points (4) and (5) shown in Fig.14.

This type of mass entrainment corresponds to the sheet stripping mechanism, while its features are visible in the flow modes at We numbers equal to 208, 360, and 650. In the modes with $We > 2000$, the droplet morphology is similar to the previous examples, but the mass entrainment occurs not only from the edges of the two main waves. At $We = 2260$, the presence of breakup products in front of the windward surface of the droplet was noted (Fig.13b). These products originated due to the wave crest stripping, though waves were of a different scale and nature. These are Kelvin-Helmholtz instability waves, and the pieces of evidence of their stripping shown in Fig.13b are perhaps the only evidence of the change in breakup mechanism within this range of modes described in [14] and obtained in the calculations.

Figure 15 presents the calculated velocity field near the droplet at $We = 2200$, from which the structures are visible same as at $We = 200$, namely, the flow separation region (*1*) at the point with maximum velocity (red region) and minimum pressure, a solitary toroidal vortex (*2*), and the reverse flow on the wake axis (*3*). At the time point equal to $70 \mu s$, a scheme of the bottom flow separation into two vortices (*1*) and (*2*) is shown that was described above for the solid model. The evidence of structures (*1*) and (*2*) are visible until the interaction time reaches $\approx 120 \mu s$, while further the flow in the wake loses its axial symmetry and disintegrates into small non-stationary

structures. They form a large stagnant area with a low average velocity comparable to the velocity in the flow deceleration zone in front of the windward surface of the droplet.

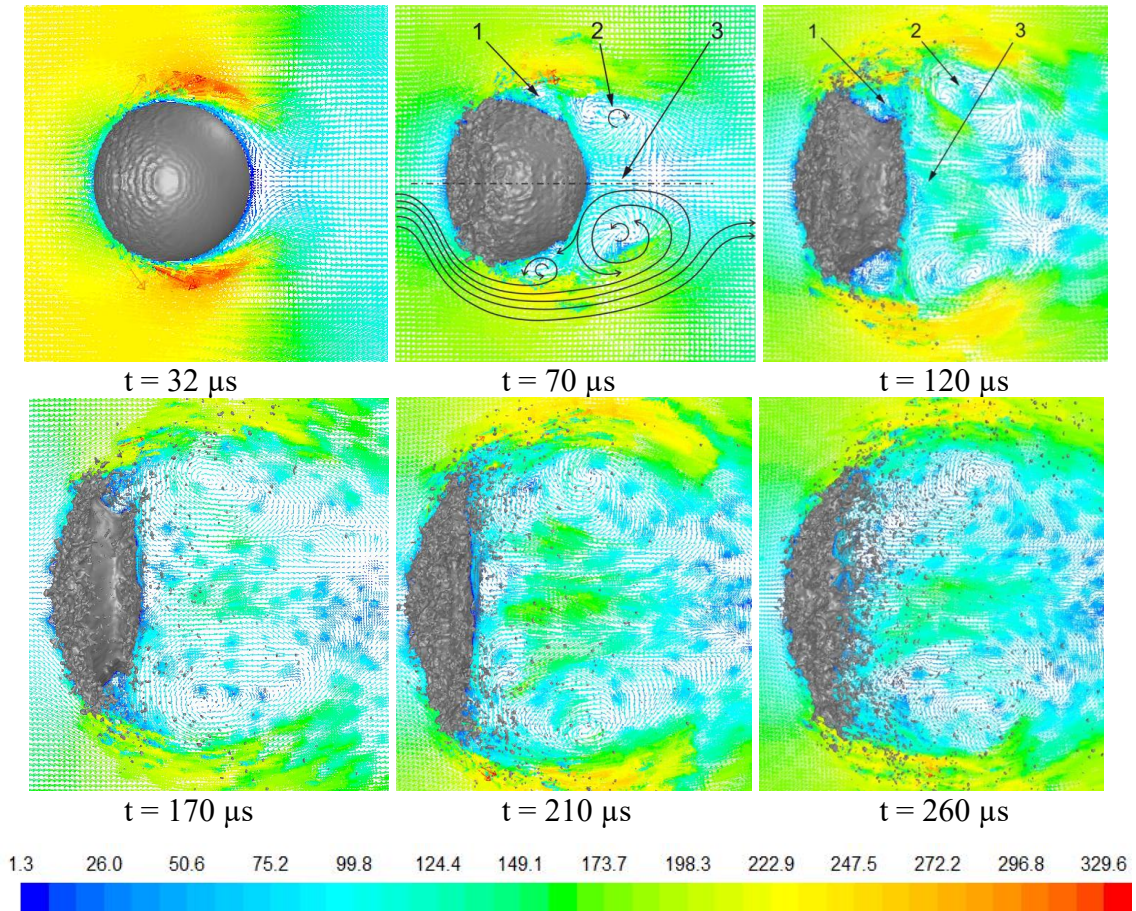


Fig. 15. Instantaneous flow pattern at different time points in streamlining of the droplet at $We = 2200$; 1 – toroidal vortex in the flow separation region, 2 – solitary toroidal vortex, 3 – reverse flow near the wake axis.

Thus, the results of the flow field simulation near the deformable droplet are in a good agreement with the PIV data for the solid droplet model at a close stage of deformation and provide a basis for constructing a phenomenological picture of the droplet's behavior in the flow behind the shock wave.

Conclusions

Experimental investigation and numerical simulation of water droplets interaction with the flow behind the passing shock wave at the Mach numbers of the shock wave $Ms = 1.109 - 1.34$ within the range of Weber numbers $We = 200 - 2200$ were performed. A comparison of the numerical simulation results with experimental data on the droplets morphology, deformation dynamics, and time of mass entrainment induction for all modes has shown a good agreement of calculations with experiments at the characteristic stages of the interaction. The flow structure near the droplet and in the droplet wake was studied at various stages of deformation, as well as

the droplet streamlining features were considered that defines the type of droplet shape evolution and the nature of mass entrainment, typical to these regimes.

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